

Finite Element Analysis of Two-Wheeler Disc Brake for Structural Optimization

M. Ahmed Khalid*

*Ph.D. Candidate, Department of Mechanical Engineering, King Fahd University of Petroleum and Minerals, Dhahran, Saudi Arabia

ABSTRACT

A piece of single framework has been considered and created in request to meet safe necessity. Rather than having air sack, great suspension frameworks, great taking care of safe cornering, there is one most basic framework in the vehicle which is slowing mechanism in the vehicle will put a traveler in dangerous position. In this way, it is must for all vehicles to have legitimate stopping mechanism. It is must for all vehicles to have appropriate slowing mechanism. In this paper carbon ironstone grid Disc brake material use for figuring typical power, shear power and cylinder drive. And also ascertaining the brake separation of Disc brake. The standard Disc brake bikes demonstrate utilizing as a part of Ansys and done the Thermal investigation and Modal examination likewise ascertain the diversion and Heat motion, Temperature of disc brake display. This is critical to comprehend activity power and erosion constrain on the Disc brake of new material, how disc brake works all the more productively, which can decrease the mischance that may occur in every day.

Keywords: Disc brake, Ansys, Structural Design, Brake Torque, Fem.

I. INTRODUCTION

The most imperative piece of a vehicle is Brake framework. Brakes are required to stop the vehicle inside the conceivable separation and it is finished by changing over active vitality of the vehicle into warm vitality by rubbing which is dispersed into air. The brakes are sufficiently solid to stop the vehicle inside the minimum conceivable separation. Brakes ought to likewise be reliable with security. The driver ought to have a decent control over the vehicle amid freeze braking. Amid the frenzy brake the vehicle ought not slip. The brakes ought to have appropriate antifade attributes and their adequacy ought not diminish with application. A disc brake get together comprises of Disc rotor that pivots with the wheel, Caliper gathering connected to the guiding knuckle, Disc cushions that are mounted to the caliper get together.

This work demonstrates a heat phase and dissemination of a Disc brake of a vehicle amid crisis braking and the accompanying discharge time frame. Brakes which moderate the vehicle and accordingly changes active vitality into warm vitality which brings about heating of the brake Disc. To stop the wheel, grinding material as brake cushions, mounted on a gadget called a brake caliper, is constrained mechanically, using forced water, pneumatically or electromagnetically against the two sides of the disc. Erosion makes the disc and connected wheel moderate or stop. Brakes change over movement to warm, and if the brakes get excessively hot, they turn out to be less successful, a marvel known as brake blur.

Disc style brakes advancement and utilize started in England in the 1890s. The principal caliper-type vehicle Disc brake was licensed by Frederick William Lanchester in his Birmingham, UK processing plant in 1902 and utilized effectively on Lanchester autos. Compared with drum brakes, disc brakes offer better ceasing execution, in light of the fact that the disc is all the more promptly cooled. A Disc brake comprises of a cast iron disc rushed to the wheel center and a stationary lodging called caliper. The caliper is associated with some stationary piece of the vehicle like the hub packaging or the stub pivot as is shown in two sections each part containing a cylinder. In the middle of every cylinder and the disc there is an erosion cushion held in position by holding pins, spring plates and so forth. Entries are jaded in the caliper for the liquid to enter or leave each lodging. The entries are additionally associated with another for dying. Every chamber contains elastic fixing ring between the chamber and cylinder. A schematic graph is appeared in the figure 1.

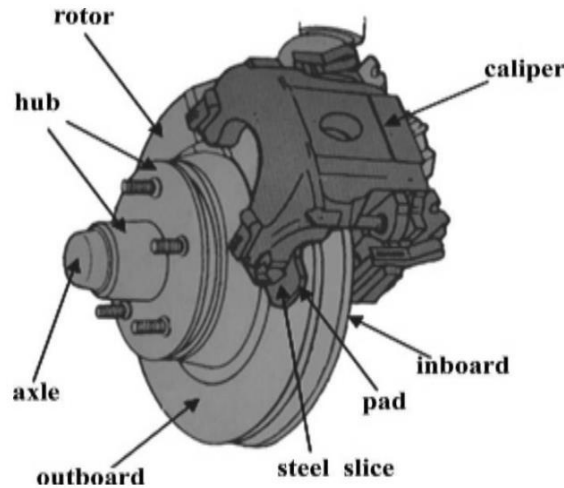


Fig 1 Computer 3D design of Disk Brake

II. PROBLEM OCCURRED IN DISC BRAKE

Discs are made up primarily dark cast iron, so Discs are harmed in one of three ways: scarring, breaking, distorting or inordinate rusting. Administration shops will some of the time react to any Disc issue by changing out the discs completely. This is done for the most part where the cost of another Disc may really be lower than the cost of laborers to restore the first disc. Mechanically this is superfluous unless the Disc have achieved maker's base prescribed thickness, which would make it perilous to utilize them, or vane rusting. Most driving vehicle producers prescribe brake Disc skimming (US: turning) as an answer for sidelong run-out, vibration issues and brake commotions. The machining procedure is performed in a brake machine, which evacuates a thin layer off the Disc surface to tidy up minor harm and restore uniform thickness. Machining the Disc as essential will increase the way out of the present discs on the vehicle. Slowing mechanisms depend on grating to convey the vehicle to an end – water driven weight pushes brake cushions against a cast iron disc or brake shoes against within a cast iron drum. At the point when a vehicle is decelerated, stack is exchanged to the front wheels – this implies the front brakes do a large portion of the work in halting the vehicle. Scarring can happen if brake cushions are not changed immediately when they achieve the finish of their administration life and are viewed as exhausted.

Breaking is constrained generally to bored plates, which may grow little splits around edges of gaps bored close to the edge of the disc because of the circle's uneven rate of extension in serious obligation conditions. The discs are ordinarily produced using cast media and a specific measure of what is known as "surface rust" is ordinary. In some cases, a noisy clamor or sharp screech happens when the brakes are connected. Most brake screech is created by vibration (reverberation shakiness) of the brake segments, particularly the cushions and discs (known as power coupled excitation). This kind of screech ought not contrarily influence brake halting execution.

III. CALCULATION OF DISC BRAKE

The powers following up on the inward and external rotor faces because of contact with brake cushions are appeared in Figure 2.

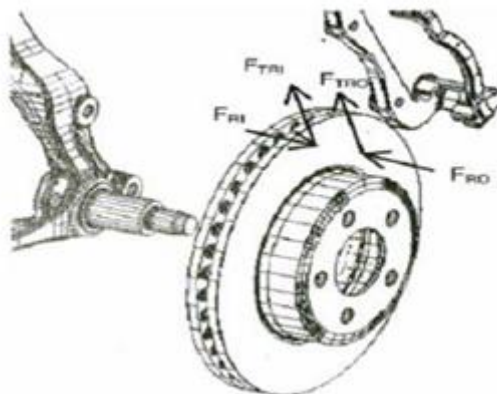


Fig 2 Forces Acting on Rotor Due to Contact with Brake Pads

Rotor plate measurement = 239mm($239 \times 10^{-3}m$)

Rotor disc material = carbon ceramic matrix

Pad break territory = 1999mm² ($1999 \times 10^{-6}m$)

Pad break material = Asbestos

Coefficient of grinding(Wet) = 0.04 – 0.12

Coefficient of erosion(Dry) = 0.2 – 0.4

Maximum Temperature = 3490_c

Maximum Weight = 1Mpa(106pa)

Tangential power among cushion and rotor (inner face), FTRI

$$FTRI = u1.FRI$$

Where, FTRI = Normal power between cushion break and rotor(inner)

u1 = Coefficient of grinding = 0.3

$$FRI = \frac{P_{max}}{2} \times A \text{ cushion territory}$$

so, FTRI = u1.FRI

$$FTRI = (0.3)(0.3) \left(1 \times \frac{106N}{m^2} \right) (1999 \times 106m^2)$$

$$FTRI = 500N$$

Distracting power amongst cushion and rotor (external face), FTRO. In this FTRO rise to FTRI on the grounds that same ordinary power and same material

Brake Torque (TB) –

With the suspicion of equivalent coefficients of grating and ordinary powers FR on the inward and external appearances:

$$TB = FTR$$

Where TB = Brake torque

u = Coefficient of contact

FT = Total typical powers on Disc brake
= FTRI + FTRO

$$FT = 1000N$$

R = Radius of rotor plate

In this way, TB = (1000)(119×10^{-3})

$$TB = 119Nm$$

Brake Distance (x) –

We realize that unrelated braking power acting at the purpose of contact of the brake, and

Work done = FT.x (Equation A)

Where FT = FTRI + FTRO

X = Distance travelled(in meter) by the vehicle before it stops.

We know motor vitality of the vehicle

$$\text{Motor vitality} = \frac{(mv^2)}{2} \dots \dots \dots (\text{Equation B})$$

Where m = Mass of the vehicle

v = Velocity of vehicle

Keeping in mind the end goal to convey the vehicle to rest, the work done against grinding must be equivalent to active vitality of the vehicle. Thusly comparing (Equation A) and (Equation B)

$$FT.x = \frac{mv^2}{2}$$

$$\text{Assumption } v = \frac{100kg}{h} = 27.772 \frac{m}{s}$$

M = 131kg(Dry weight of vehicle)

$$\text{so we get } x = \frac{mv^2}{2FT}$$

$$x = \frac{131 \times 27.772}{(2 \times 1000)} m$$

$$x = 50.81m$$

$$\text{Heat Generation}(Q) = M \cdot c_p \cdot \Delta T \frac{j}{s}$$

$$\text{Flue}(q) = \frac{Q}{A} \frac{w}{m^2}$$

$$\text{Thermal Gradient}(K) = q \frac{k}{m}$$

Carbon Carmic Matrix –

$$\text{Heat produced } Q = m \cdot c_p \cdot \Delta T$$

$$\text{Mass of plate} = 0.4 \text{ k}$$

$$\text{Specific Heat Capacity} = 798 \frac{j}{kg} o_c$$

$$\text{Time taken stopping the vehicle} = 5 \text{ sec}$$

$$\text{Developed Temperature Contrast} = 130_c$$

$$Q = 0.4 \cdot 798 \cdot 13 = 5998 \text{ j}$$

$$\text{Area of Disc} = \pi \times (R^2 - r^2) = \pi(0.119^2 - 0.053^2) = 0.03566 \text{ m}^2$$

$$\text{Heat flux} = \frac{\text{Heat Generated}}{\frac{\text{Second}}{\text{zone}}} = \frac{5998}{0.03566} = 33.639 \frac{Kw}{m^2}$$

$$\text{Thermal Gradient} = \frac{\text{Heat Flux}}{\text{Thermal Conductivity}}$$

$$= 33.639 \cdot 10 \frac{3}{40}$$

$$= 840.97 \text{ k/m}$$

IV. FEM USING ANSYS

Ansys is one of the valuable programming for outline investigation in mechanical building. This product depends on the Finite Element Method (FEM) to recreate the working states of your plans and anticipate their conduct. FEM requires the arrangement of larges frameworks of conditions. Controlled by quick solvers, Ansys makes it feasible for planners to rapidly check the integrity of their outlines and look for the ideal arrangement. An item improvement cycle ordinarily incorporates the accompanying advances:

- Build your model in the Pro-Engineer framework.
- Prototype the plan.
- Test the model in the field.
- Evaluate the consequences of the field tests.
- Modify the outline in view of the field test comes about.

Model of Disk Brake

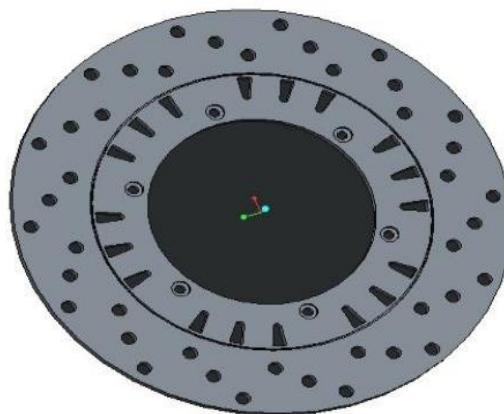


Fig 3 Model of Disc Brake

Thermal Analysis of Disk Brake

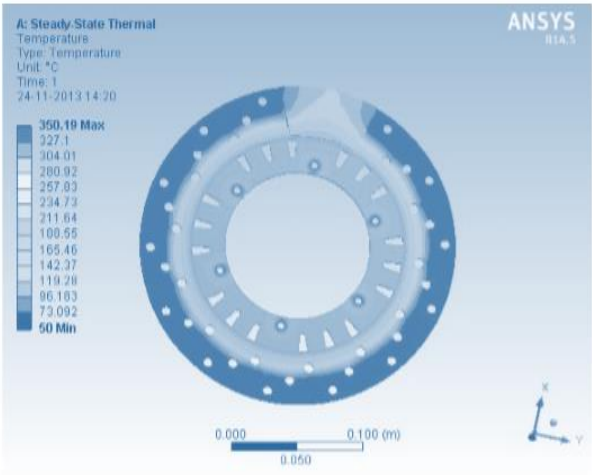


Fig 4 Temperature Distribution

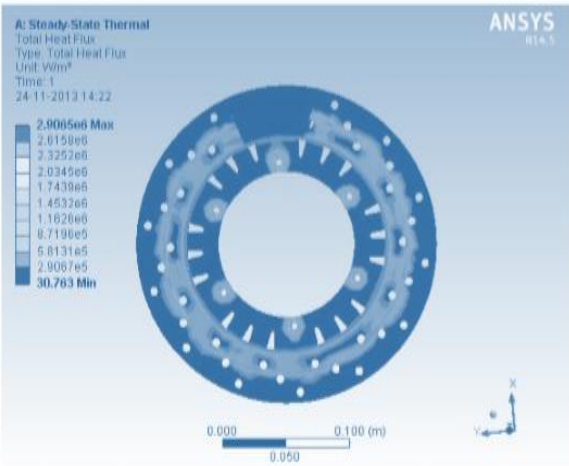


Fig 5 Total Heat Flux

Model Analysis of Disc Brake

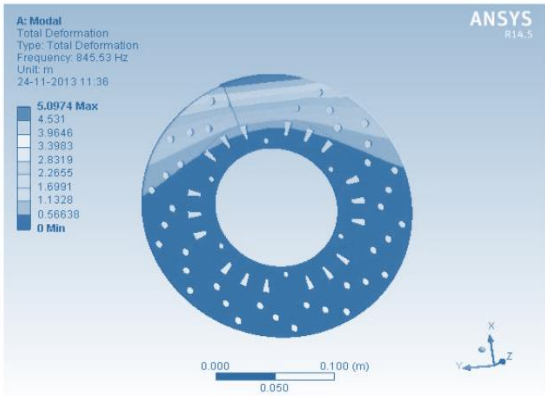


Fig 6 Frequency and Total Deformation of Disc Brake

V. CONCLUSIONS

Utilizing carbon earthenware network circle brake material ascertaining typical power, shear power and cylinder compel and furthermore figuring the brake separation of Disc brake. The standard disc brake bikes demonstrate utilizing as a part of Ansys, the Thermal and Modal Analysis ascertain the avoidance, add up to warm transition, Frequency and temperature of Disc brake shows. This is essential to comprehend activity power and erosion constrain on the disc brake new material, which utilize disc brake works all the more effectively, which can decrease the mischance that may occur in every day.

VI. ACKNOWLEDGMENT

I would like to thanks KLUniversity for providing ANSYS tool.

VII. REFERENCES

- [1] Ameer Shaik and Lakshmi Srinivas, "Structural and Thermal Analysis of Disc Brake Without Cross-drilled Rotor of Race Car", 'International Journal of Advanced Engineering Research and Studies', 2012, Vol. I, Issn 2249-8974, pp 39-43.
- [2] Chogdu Cho and Sooick Ahn, "Thermo-Elastic Analysis for Chattering Phenomenon of Automotive Disk Brake", 'KSME International Journal', 2001, Vol-15, pp 569-579.
- [3] B. N and P. G, "Design and Analysis of Disc Brake Rotor for a Two-wheeler," International Journal of Mechanical and Industrial Technology, vol. 1, no. 1, pp. 7-12, 2014.
- [4] H. S. Haripal Singh, "Thermal Analysis of Disc Brake Using COMSOL," International Journal on Emerging Technologies, vol. 3, no. 1, pp. 84-88, 2012.
- [5] B. M. Belhocine A, "Temperature and Thermal Stresses of Vehicles Gray Cast Brake," Journal of Applied Research and Technology, vol. 11, pp. 675-682, 2013.
- [6] Seelam Vasavi Sai Viswanada Prabhu Deva Kumar, Shyam Akashe, Vikram Kumar "Advanced Control of Switching Ignition by Smart Helmet", I.J. Image, Graphics and Signal Processing, 2018, 2, 34-42.
- [7] D. Swapnil R. Abhang, "Design and Analysis of Disc Brake," International Journal of Engineering Trends and Technology, vol. 8, pp. 165-, 2014.
- [8] Guru Murthy Nathi, K. Gowtham and Satish Reddy, "Coupled Structural / Thermal Analysis of Disc Brake", IJRET 2012, Vol.1, pp.539-553.
- [9] M. T. V and D. S. P. M, "Structural and Thermal Analysis of Rotor Disc of Disc Brake," International Journal of Innovative Research in Science, Engineering and Technology, vol. 2, no. 12, pp. 7741-7749, 2013.